

PROGRAM NAME: ETABS
 REVISION NO.: 0

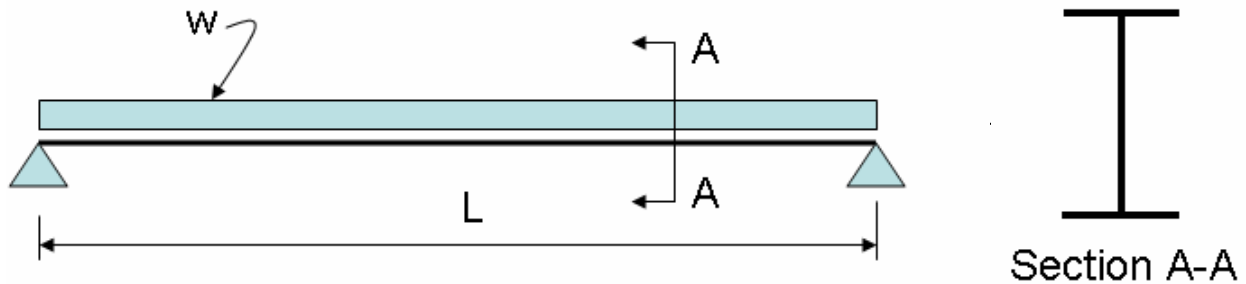
KBC 2009 Example 001

WIDE FLANGE MEMBER UNDER BENDING

EXAMPLE DESCRIPTION

The design flexural strengths are checked for the beam shown below. The beam is loaded with a uniform load of 6.5 kN/m (D) and 11 kN/m (L). The flexural moment capacity is checked for three unsupported lengths in the weak direction, $L_b = 1.75$ m, 4 m and 12 m.

GEOMETRY, PROPERTIES AND LOADING



Member Properties

W460x74
 $E = 205,000$ MPa
 $F_y = 345$ MPa

Loading

$w = 6.5$ kN/m (D)
 $w = 11.0$ kN/m (L)

Geometry

Span, $L = 12$ m

TECHNICAL FEATURES TESTED

- Section Compactness Check (Bending)
- Member Bending Capacities
- Unsupported length factors

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RESULTS COMPARISON

Independent results are comparing with the results of ETABS.

Output Parameter	ETABS	Independent	Percent Difference
Compactness	Compact	Compact	0.00%
$C_b (L_b = 1.75\text{m})$	1.004	1.002	0.20%
$\phi_b M_n (L_b = 1.75\text{m})$ (kN-m)	515.43	515.43	0.00%
$C_b (L_b = 4\text{m})$	1.015	1.014	0.10%
$\phi_b M_n (L_b = 4\text{m})$ (kN-m)	394.8	394.2	0.15%
$C_b (L_b = 12\text{m})$	1.136	1.136	0.00%
$\phi_b M_n (L_b = 12\text{m})$ (kN-m)	113.48	113.45	0.03%

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CONCLUSION

The results show an acceptable comparison with the independent results.

HAND CALCULATION

Properties:

Material:

$$E = 205,000 \text{ MPa}, F_y = 345 \text{ MPa}$$

Section: W460x74

$$b_f = 191 \text{ mm}, t_f = 14.5 \text{ mm}, d = 457 \text{ mm}, t_w = 9 \text{ mm}$$

$$h = d - 2t_f = 457 - 2 \cdot 14.5 = 428 \text{ mm}$$

$$h_0 = d - t_f = 457 - 14.5 = 442.5 \text{ mm}$$

$$S_{33} = 1457.3 \text{ cm}^3, Z_{33} = 1660 \text{ cm}^3$$

$$I_y = 1670 \text{ cm}^4, r_y = 42 \text{ mm}, C_w = 824296.4 \text{ cm}^6, J = 51.6 \text{ cm}^4$$

$$r_{ts} = \sqrt{\frac{\sqrt{I_y C_w}}{S_{33}}} = \sqrt{\frac{\sqrt{1670 \cdot 824296.4}}{1457.3}} = 50.45 \text{ mm}$$

$$R_m = 1.0 \text{ for doubly-symmetric sections}$$

Other:

$$c = 1.0$$

$$L = 12 \text{ m}$$

Loadings:

$$w_u = (1.2w_d + 1.6w_l) = 1.2(6.5) + 1.6(11) = 25.4 \text{ kN/m}$$

$$M_u = \frac{w_u L^2}{8} = \frac{25.4 \cdot 12^2}{8} = 457.2 \text{ kN-m}$$

Section Compactness:

Localized Buckling for Flange:

$$\lambda = \frac{b_f}{2t_f} = \frac{191}{2 \cdot 14.5} = 6.586$$

$$\lambda_p = 0.38 \sqrt{\frac{E}{F_y}} = 0.38 \sqrt{\frac{205,000}{345}} = 9.263$$

$\lambda < \lambda_p$, No localized flange buckling

Flange is Compact.

Localized Buckling for Web:

$$\lambda = \frac{h}{t_w} = \frac{428}{9} = 47.56$$

$$\lambda_p = 3.76 \sqrt{\frac{E}{F_y}} = 3.76 \sqrt{\frac{205,000}{345}} = 91.654$$

$\lambda < \lambda_p$, No localized web buckling

Web is Compact.

Section is Compact.

Section Bending Capacity:

$$M_p = F_y Z_{33} = 345 \cdot 1660 = 572.7 \text{ kN-m}$$

Lateral-Torsional Buckling Parameters:

Critical Lengths:

$$L_p = 1.76 r_y \sqrt{\frac{E}{F_y}} = 1.76 \cdot 42 \sqrt{\frac{205,000}{345}} = 1801.9 \text{ mm} = 1.8 \text{ m}$$

$$L_r = 1.95 r_{ts} \frac{E}{0.7 F_y} \sqrt{\frac{Jc}{S_{33} h_o}} \sqrt{1 + \sqrt{1 + 6.76 \left(\frac{0.7 F_y S_{33} h_o}{E Jc} \right)^2}}$$

$$L_r = 1.95 \cdot 50.45 \frac{205,000}{0.7 \cdot 345} \sqrt{\frac{51.6 \cdot 1}{1457.3 \cdot 44.25}} \sqrt{1 + \sqrt{1 + 6.76 \left(\frac{0.7 \cdot 345 \cdot 1457.3 \cdot 44.8}{205,000 \cdot 51.6 \cdot 1} \right)^2}}$$

$$L_r = 5.25 \text{ m}$$

Non-Uniform Moment Magnification Factor:

For the lateral-torsional buckling limit state, the non-uniform moment magnification factor is calculated using the following equation:

$$C_b = \frac{12.5M_{\max}}{2.5M_{\max} + 3M_A + 4M_B + 3M_C} R_m \leq 3.0 \quad \text{Eqn. 1}$$

Where M_A = first quarter-span moment, M_B = mid-span moment, M_C = second quarter-span moment.

The required moments for Eqn. 1 can be calculated as a percentage of the maximum mid-span moment. Since the loading is uniform and the resulting moment is symmetric:

$$M_A = M_C = 1 - \frac{1}{4} \left(\frac{L_b}{L} \right)^2$$

Member Bending Capacity for $L_b = 1.75 \text{ m}$:

$$M_{\max} = M_B = 1.00$$

$$M_A = M_C = 1 - \frac{1}{4} \left(\frac{L_b}{L} \right)^2 = 1 - \frac{1}{4} \left(\frac{1.75}{12} \right)^2 = 0.995$$

$$C_b = \frac{12.5(1.00)}{2.5(1.00) + 3(0.995) + 4(1.00) + 3(0.995)}$$

$$C_b = 1.002$$

$L_b < L_p$, Lateral-Torsional buckling capacity is as follows:

$$M_n = M_p = 572.7 \text{ kN-m}$$

$$\phi_b M_n = 0.9 \cdot 572.7$$

$$\phi_b M_n = 515.43 \text{ kN-m}$$

Member Bending Capacity for $L_b = 4$ m:

$$M_{\max} = M_B = 1.00$$

$$M_A = M_C = 1 - \frac{1}{4} \left(\frac{L_b}{L} \right)^2 = 1 - \frac{1}{4} \left(\frac{4}{12} \right)^2 = 0.972$$

$$C_b = \frac{12.5(1.00)}{2.5(1.00) + 3(0.972) + 4(1.00) + 3(0.972)}$$

$$C_b = 1.014$$

$L_p < L_b < L_r$, Lateral-Torsional buckling capacity is as follows:

$$M_n = C_b \left[M_p - (M_p - 0.7F_y S_{33}) \left(\frac{L_b - L_p}{L_r - L_p} \right) \right] \leq M_p$$

$$M_n = 1.014 \left[572.7 - (572.7 - 0.7 \cdot 0.345 \cdot 1457.3) \left(\frac{4.00 - 1.80}{5.25 - 1.80} \right) \right] = 437.97 \text{ kN-m}$$

$$\phi_b M_n = 0.9 \cdot 437.97$$

$$\phi_b M_n = 394.2 \text{ kN-m}$$

Member Bending Capacity for $L_b = 12$ m:

$$M_{\max} = M_B = 1.00$$

$$M_A = M_C = 1 - \frac{1}{4} \left(\frac{L_b}{L} \right)^2 = 1 - \frac{1}{4} \left(\frac{12}{12} \right)^2 = 0.750$$

$$C_b = \frac{12.5(1.00)}{2.5(1.00) + 3(0.750) + 4(1.00) + 3(0.750)} (1.00)$$

$$C_b = 1.136$$

$L_b > L_r$, Lateral-Torsional buckling capacity is as follows:

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$$F_{cr} = \frac{C_b \pi^2 E}{\left(\frac{L_b}{r_{ts}}\right)^2} \sqrt{1 + 0.078 \frac{Jc}{S_{33} h_o} \left(\frac{L_b}{r_{ts}}\right)^2}$$

$$F_{cr} = \frac{1.136 \cdot \pi^2 \cdot 205,000}{\left(\frac{12000}{50.45}\right)^2} \sqrt{1 + 0.078 \frac{51.6 \cdot 1}{1457.3 \cdot 44.25} \left(\frac{12000}{50.45}\right)^2} = 86.5 \text{ MPa}$$

$$M_n = F_{cr} S_{33} \leq M_p$$

$$M_n = 86.5 \cdot 1457.3 = 126.056 \text{ kN-m}$$

$$\phi_b M_n = 0.9 \cdot 126.056$$

$$\phi_b M_n = 113.45 \text{ kN-m}$$