



API RP 2A-WSD 22nd Steel Frame Design Manual



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1 Introduction

This manual describes the steel frame design algorithms in the software for API Recommended Practice 2A-WSD 22nd Edition (American Petroleum Institute, 2014). The design algorithms in the software for API RP 2A-WSD 22nd cover allowable stress checks for typical structural elements used in offshore steel structures, as detailed in this manual. Such elements are tubular members and tubular joints. For other types of structural elements, the software uses AISC ASD 9th Edition. Requirements of the code not documented in this manual should be considered using other methods.

This manual documents the design details for cylindrical sections having thickness $t \ge 6$ mm, D/t < 300. Members of other section shapes are designed in accordance with AISC ASD 9th Edition (American Institute of Steel Construction, 1989).

It is important to read this entire manual before using the design algorithms to become familiar with any limitations of the algorithms or assumptions that have been made.

1.1 Units

The API RP 2A-WSD design code is based on metric and imperial units. This manual is written using imperial units, unless noted otherwise. Any units, imperial, metric, or MKS may be used in the software in conjunction with API RP 2A-WSD design.

1.2 Axes Notation

The software analysis results refer to the member local axes system, which consists of the 2-2 axis and the 3-3 axis. The API RP 2A-WSD design code refers to x-x and y-y axes, which are equivalent to the software 3-3 and 2-2 axes, respectively. These notations may be used interchangeably in the design algorithms, although every effort has been made to use the design code convention where possible.

1.3 Symbols

The following table provides a list of the symbols used in this manual, along with a short

Units 1

description. Where possible, the same symbol from the design code is used in this manual.

A Cross sectional area, in²

C Critical elastic buckling coefficient

C_h Critical hoop buckling coefficient

 C_m Reduction factor

D Outside diameter, in

E Young's modulus of elasticity, ksi

 f_a Design tensile stress, ksi

 F_a Allowable compressive stress, ksi

f_b Design bending stress, ksi

 F_b Allowable bending stress, ksi

 F_e Euler stress, ksi

 f_h Hoop stress due to hydrostatic pressure, ksi

 F_{hc} Critical hoop buckling stress, ksi

 F_{he} Elastic hoop buckling stress, ksi

 F_t Allowable tensile stress, ksi

 f_{ν} Design beam shear stress, ksi

 F_{ν} Allowable beam shear stress, ksi

 f_{vt} Design torsional shear stress, ksi

 F_{vt} Allowable torsional shear stress, ksi

 f_x Design compressive stress, ksi

 F_{xc} Inelastic local buckling stress, ksi

 F_{xe} Elastic local buckling stress, ksi

 F_{ν} Yield strength, ksi

g Gap distance, in

 I_p Polar moment of inertia, in⁴

K Effective length factor

l Unbraced length, in

L Length between stiffening rings, diaphragms, or end connections, in

M Bending moment, kip-in

M Geometric parameter

 M_a Allowable brace bending moment, kip-in

 M_t Torsional moment, kip-in

p Hydrostatic pressure, ksi

Symbols 2

P	Axial force, kip
P_a	Allowable brace axial load, kip
Q_f	Chord load factor
Q_g	Gap factor
Q_u	Ultimate strength factor
Q_eta	Geometric factor
r	Radius of gyration, in
SF_b	Safety factor for bending
SF_h	Safety factor against hydrostatic collapse
SF_x	Safety factor for axial force
t	Wall thickness, in
V	Transverse shear force, kip
ν	Poisson's ratio
heta	Angle between the chord and the brace

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2 Member Design

This chapter provides the details of the structural steel design and stress check algorithms that are used for cylindrical member design and checking at each output station in accordance with API RP 2A-WSD.

Cylindrical members subjected solely to axial tension, axial compression, bending, shear, or hydrostatic pressure are designed in accordance with API RP 2A-WSD Sections 6.2.1 to 6.2.5, respectively. Cylindrical members subjected to combined loads without hydrostatic pressure are designed in accordance with API RP 2A-WSD Sections 6.3.2 and 6.3.3. Cylindrical members subjected to combined loads with hydrostatic pressure are designed in accordance with API RP 2A-WSD Sections 6.3.4 and 6.3.5.

2.1 Safety Factors

The safety factors used in calculating allowable stresses in the following sections are defined as:

Loading **Design Condition Axial** Axial **Bending** Hoop **Tension** Compression Compression 1.67 2.0 F_y/F_b 2.0 Basic allowable stresses One-third increase in allowable 1.25 1.5 $F_{v}/(1.33F_{b})$ 1.5 stresses is permitted

Table 1 - Safety factors

2.2 Tension Check

Members subjected to axial tension are checked for the following condition:

$$\frac{f_a}{F_t} \le 1.0$$
 [API 6.2.1]

The allowable tensile stress, F_t , is defined as:

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$$F_t = 0.6F_v$$
 [API Eq. 6.1]

2.3 Compression Check

Members subjected to axial compression are checked for the following condition:

$$\frac{f_a}{F_a} \le 1.0$$
 [API 6.2.2]

The allowable compressive stress, F_a , is defined as:

$$F_{a} = \begin{cases} \frac{\left[1 - \frac{(Kl/r)^{2}}{2C_{c}^{2}}\right] F_{y}}{5/3 + \frac{3(Kl/r)}{8C_{c}} - \frac{(Kl/r)^{3}}{8C_{c}^{3}}} & \text{for } Kl/r < C_{c} \\ \frac{12\pi^{2}E}{23(Kl/r)^{2}} & \text{for } Kl/r \ge C_{c} \end{cases}$$
[API Eq. 6.2 & Eq. 6.3]

where,

$$C_c = \left[\frac{2\pi^2 E}{F_y}\right]^{0.5}$$

$$F_y = \begin{cases} F_y & \text{for } D/t \le 60\\ \min(F_{xe}, F_{xc}) & \text{for } D/t > 60 \end{cases}$$

For members with D/t > 60, the yield strength, F_y , in the above equations is replaced by the critical local buckling stress, defined as the minimum of F_{xe} or F_{xc} .

The elastic local buckling stress, F_{xe} , is defined as:

$$F_{xe} = 2CE t/D$$
 [API Eq. 6.4]

where the critical elastic buckling coefficient, C = 0.3.

The inelastic local buckling stress, F_{xc} , is defined as:

$$F_{xc} = F_y [1.64 - 0.23(D/t)^{1/4}] \le F_{xe}$$
 [API Eq. 6.5]

2.4 Flexure Check

Members subjected to bending are checked for the following condition:

$$\frac{f_b}{F_b} \le 1.0$$
 [API 6.2.3]

Compression Check 5

The allowable bending stress, F_b , is defined as:

$$F_b = \begin{cases} 0.75F_y & \text{for } \frac{D}{t} \le \frac{1500}{F_y} \\ \left[0.84 - 1.74 \frac{F_y D}{Et} \right] F_y & \text{for } \frac{1500}{F_y} < \frac{D}{t} \le \frac{3000}{F_y} \end{cases}$$
 [API Eq. 6.6, 6.7, and 6.8]
$$\left[0.72 - 0.58 \frac{F_y D}{Et} \right] F_y & \text{for } \frac{3000}{F_y} < \frac{D}{t} \le 300$$

2.5 Shear Check

Members subjected to beam shear are checked for the following condition:

$$\frac{f_v}{F_v} \le 1.0$$
 [API 6.2.4.1]

The maximum beam shear stress, f_v , and the allowable beam shear stress F_v are defined as:

$$f_v = \frac{V}{0.5A}$$
 [API Eq. 6.9]

$$F_v = 0.4F_y$$
 [API Eq. 6.10]

Members subjected to torsional shear are checked for the following condition:

$$\frac{f_{vt}}{F_{vt}} \le 1.0$$
 [API 6.2.4.2]

The maximum torsional shear stress, f_{vt} , and the allowable torsional shear stress F_{vt} are defined as:

$$f_{vt} = \frac{M_t(D/2)}{I_p}$$
 [API Eq. 6.11]

$$F_{vt} = 0.4F_y$$
 [API Eq. 6.12]

2.6 Hoop Buckling Check

Members subjected to external pressure are checked for the following condition:

$$f_h \le F_{hc}/SF_h$$
 [API Eq. 6.13]

The hoop stress due to hydrostatic pressure, f_h , is defined as:

$$f_h = pD/2t [API Eq. 6.14]$$

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The critical hoop buckling stress, F_{hc} , is defined as:

$$F_{hc} = \begin{cases} F_{he} & \text{for} & F_{he} \le 0.55F_y \\ 0.45F_y + 0.18F_{he} & \text{for} & 0.55F_y < F_{he} \le 1.6F_y \\ \frac{1.31F_y}{1.15 + (F_y/F_{he})} & \text{for} & 1.6F_y < F_{he} \le 6.2F_y \\ F_y & \text{for} & F_{he} > 6.2F_y \end{cases}$$
[API Eq. 6.18]

The elastic hoop buckling stress, F_{he} , is defined as:

$$F_{he} = 2C_h E t/D$$
 [API Eq. 6.16]

The critical hoop buckling coefficient, C_h , is defined as:

$$C_h = \begin{cases} 0.44 \, t/D & \text{for} & M \ge 1.6 \, D/t \\ 0.44 (t/D) + \frac{0.21 (D/t)^3}{M^4} & \text{for} & 0.825 \, D/t \le M < 1.6 \, D/t \\ 0.736 / (M - 0.636) & \text{for} & 3.5 \le M < 0.825 \, D/t \\ 0.755 / (M - 0.559) & \text{for} & 1.5 \le M < 3.5 \\ 0.8 & \text{for} & M < 1.5 \end{cases}$$

The geometric parameter, M, is defined as:

$$M = \frac{L}{D} (2D/t)^{0.5}$$
 [API Eq. 6.17]

2.7 Axial Tension and Bending Check

Members subjected to combined axial tension and bending loads, without hydrostatic pressure, are checked for the following condition:

$$\frac{f_a}{0.6F_y} + \frac{\sqrt{f_{bx}^2 + f_{by}^2}}{F_b} \le 1.0$$
 [API Eq. 6.21]

Members subjected to combined axial tension, bending, and hydrostatic pressure are checked for the following condition:

$$A^2 + B^2 + 2\nu |A|B \le 1.0$$
 [API Eq. 6.26]

where,

$$A = \frac{f_a + f_b - (0.5f_h)}{F_{\nu}} (SF_{\nu})$$

$$B = \frac{f_h}{F_{hc}}(SF_h)$$

2.8 Axial Compression and Bending Check

Members subjected to combined axial compression and bending, without hydrostatic pressure, are checked for the following conditions:

$$\frac{f_a}{F_a} + \frac{\sqrt{\left[\frac{C_{mx}f_{bx}}{1 - \frac{f_a}{F_{ex}'}}\right]^2 + \left[\frac{C_{my}f_{by}}{1 - \frac{f_a}{F_{ey}'}}\right]^2}}{F_b} \le 1.0$$
 [API Eq. 6.23]

$$\frac{f_a}{0.6F_y} + \frac{\sqrt{f_{bx}^2 + f_{by}^2}}{F_b} \le 1.0$$
 [API Eq. 6.21]

where,

$$F'_e = \frac{12\pi^2 E}{23(Kl/r)^2}$$
 [AISC H1]

The reduction factors, C_{mx} and C_{my} are calculated according to AISC H1.

If $\frac{f_a}{F_a} \le 0.15$, the previous two conditions are substituted by the following condition:

$$\frac{f_a}{F_a} + \frac{\sqrt{f_{bx}^2 + f_{by}^2}}{F_b} \le 1.0$$
 [API Eq. 6.22]

Members subjected to combined axial compression, bending, and hydrostatic pressure are checked for the following conditions:

$$\frac{f_a + (0.5f_h)}{F_{xc}}(SF_x) + \frac{f_b}{F_v}(SF_b) \le 1.0$$
 [API Eq. 6.27]

$$SF_h \frac{f_h}{F_{hc}} \le 1.0$$
 [API Eq. 6.28]

If $f_x > 0.5F_{ha}$, the following condition is also satisfied:

$$\frac{f_x - 0.5F_{ha}}{F_{aa} - 0.5F_{ha}} + \left(\frac{f_h}{F_{ha}}\right)^2 \le 1.0$$
 [API Eq. 6.29]

where,

$$F_{aa} = \frac{F_{xe}}{SF_x}$$

$$F_{ha} = \frac{F_{he}}{SF_h}$$

$$f_x = f_a + f_b + (0.5f_h)$$

3 Joint Design

This chapter provides the details of the joint punching load check algorithm that is used for tubular joints in accordance with API RP 2A-WSD Section 7.3.

API RP 2A-WSD allows the joints to be designed on the basis of nominal loads in the braces.

3.1 Joint Geometry

Figure 1 illustrates some of the geometric parameters used in the punching load check.

- d Brace diameter, in
- D Chord diameter, in
- g Gap distance, in
- t Brace thickness, in
- T Chord thickness, in
- θ Angle measured from the chord to the brace

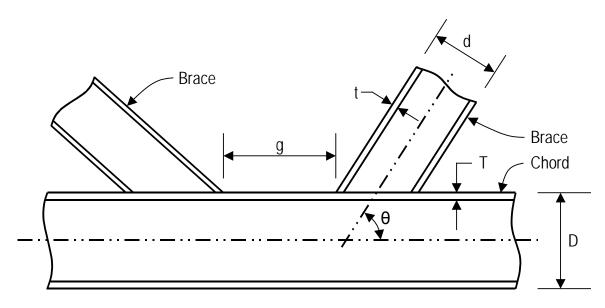


Figure 1 - Joint geometry

The following geometric parameters are derived from those in Figure 1.

$$\beta = \frac{d}{D} \quad \gamma = \frac{D}{2T} \quad \tau = \frac{t}{T}$$

3.2 Allowable Capacities

The allowable brace axial load, P_a , is defined as:

$$P_a = Q_u Q_f \frac{F_{yc} T^2}{FS \sin \theta}$$
 [API Eq. 7.1]

The allowable brace bending moment, M_a , is defined as:

$$M_a = Q_u Q_f \frac{F_{yc} T^2 d}{FS \sin \theta}$$
 [API Eq. 7.2]

where the safety factor, FS = 1.60.

The chord load factor, Q_f , is defined as:

$$Q_f = \left[1 + C_1 \left(\frac{FSP_c}{P_y}\right) - C_2 \left(\frac{FSM_{ipb}}{M_p}\right) - C_3 A^2\right]$$
 [API Eq. 7.3]

The parameter, A, is defined as:

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$$A = \left[\left(\frac{FSP_c}{P_y} \right)^2 + \left(\frac{FSM_c}{M_p} \right)^2 \right]^{0.5}$$
 [API Eq. 7.4]

where the safety factor, FS = 1.20 where the 1/3 increase is applicable. P_c is the nominal axial load and M_c is the nominal bending resultant in the chord.

$$M_c = \sqrt{M_{ipb}^2 + M_{opb}^2}$$
 [API 4.3.4]

The coefficients, C_1 , C_2 , and C_3 , are determined based on API Table 7.3.

C3 Joint Type **C1 C2** K axial 0.2 0.2 0.3 0.3 0.8 T&Y axial 0 X axial $\beta \leq 0.9$ 0.2 0.2 X axial $\beta = 1.0$ -0.2 0 0.2 0.4 0 All joints moment

Table 2 – Coefficients, C_1 , C_2 , and C_3

The ultimate strength factor, Q_u , is determined based on API Table 7.2.

	Brace Action					
Joint Class	Axial Tension	Axial Compression	In-plane Bending	Out-of-plane Bending		
K	$(16 + 1.2\gamma)\beta^{1.2}Q_g \le 40\beta^{1.2}Q_g$					
T&Y	30β	2.8 + $(20 + 0.8\gamma)\beta^{1.6}$ $\leq 2.8 + 36\beta^{1.6}$	$(5+0.7\gamma)\beta^{1.2}$	$2.5+(4.5+0.2\gamma)\beta^{2.6}$		
X	23 β for $\beta \le 0.9$ 20.7 + (β – 0.9) (17 γ – 220) for β > 0.9	[2.8 + $(12 + 0.1\gamma)\beta$] Q_{β}				

Table 3 - Factor, Qu

The geometric factor, Q_{β} , is defined as:

$$Q_{\beta} = \begin{cases} \frac{0.3}{\beta(1 - 0.833\beta)} & \text{for } \beta > 0.6\\ 1.0 & \text{for } \beta \le 0.6 \end{cases}$$
 [API Table 7.2]

The gap factor, Q_g , is defined as:

$$Q_g = \begin{cases} 1 + 0.2[1 - 2.8 \, g/D]^3 \ge 1.0 & \text{for} \quad g/D \ge 0.05 \\ 0.13 + 0.65 \phi \gamma^{0.5} & \text{for} \quad g/D < -0.05 \end{cases}$$
 [API Table 7.2]

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$$\phi = t \, F_{yb} / \big(T F_y \big)$$

3.3 Axial and Bending Check

Joints are checked for the following condition:

$$\left| \frac{P}{P_a} \right| + \left(\frac{M}{M_a} \right)_{ipb}^2 + \left| \frac{M}{M_a} \right|_{opb} \le 1.0$$
 [API Eq. 7.6]

The subscripts IPB and OPB correspond to in-plane bending and out-of-plane bending, respectively.

4 References

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